# UNIVERSITI KUALA LUMPUR <br> Malaysia France Institute 

## FINAL EXAMINATION <br> SEPTEMBER 2014 SESSION

| SUBJECT CODE | $:$ FCD20102 |
| :--- | :--- |
| SUBJECT TITLE | $:$ FLUID MECHANICS |
| LEVEL | $:$ DIPLOMA |
| TIME / DURATION | $:$(2 HOURS ) |
|  | $: 31$ DECEMBER 2014 |

INSTRUCTIONS TO CANDIDATES

1. Please read the instructions given in the question paper CAREFULLY.
2. This question paper is printed on both sides of the paper.
3. Please write your answers on the answer booklet provided.
4. Answer should be written in blue or black ink except for sketching, graphic and illustration.
5. This question paper consists of TWO (2) sections. Section A and B. Answer all questions in Section A. For Section B, answer two (2) question only.
6. Answer questions in English.
7. Fomula is appended in the Appendices.

## SECTION A (Total: 60 marks)

INSTRUCTION: Answer ALL questions.
Please use the answer booklet provided.

## Question 1

(a) What is the definition of mass?
(b) What is the definition of weight?
(c) Calculate the weight of a reservoir of oil if it has a mass of 500 kg . Answer in kN .
(4 marks)
(d) If the reservoir from question 1 (c) has a volume of $0.817 \mathrm{~m}^{3}$, find the :-
i. Density of oil.
ii. Specific weight of oil.

> (4 marks)
iii. Specific gravity of oil.
(4 marks)

## Question 2

a) What is the definition of pressure?
b) State two (2) important principles about pressure described by Pascal's Low.
(6 marks)
c) Refer to figure Q2, shows a container of liquid with a movable piston supporting a load. Determine the pressure in the liquid under the piston if the total weight of the piston and the load 500 N and the area of the piston is $1500 \mathrm{~cm}^{2}$. Final answer in MPa


Figure Q2(C): Container of liquid

## Question 3

a) Refer figure Q3, shown the manometer of oil and water. Determine the pressure difference between point $A$ and $B$ if the specific gravity of oil is 0.85
i. Find the density of oil and water.
(5 marks)
ii. Determine the pressure difference between point $A$ and $B$.
(15 marks)


Figure Q3: Differential Manometer

## SECTION B (Total: 40 marks)

## INSTRUCTION: Answer only TWO questions.

Please use the answer booklet provided.

## Question 4

Refer to Figure Q4. Chilled Water at $70^{\circ} \mathrm{C}$ is flowing at section 1 to section 2, the velocity of flow is $3.0 \mathrm{~m} / \mathrm{s}$ with its diameter is of 80 mm . Section 2 is 20 mm in diameter. Assuming there is no energy loss in the system,


Figure Q4: Conservation of Bernoulli's
(a) Calculate the velocity in $\mathrm{m} / \mathrm{s}$
(b) In order to maintain the same pressure $\mathrm{P}_{1}=\mathrm{P}_{2}$, calculate the elevation h in $\mathrm{N} . \mathrm{m} / \mathrm{N}$.

## Question 5

The volume flow rate through the pump shown in Figure Q5, is $0.014 \mathrm{~m}^{3} / \mathrm{s}$. The fluid being pump is oil with a specific gravity of 0.85 . Calculate the energy delivered by the pump to the oil per unit weight of oil flowing in the system. Neglect any energy losses in the system.
i. Find the specific weight of oil.
ii. Determine the energy added to the system in SI unit.
iii. Determine the power delivered by the pump to the oil in SI unit


Figure Q5: Power of the pump

## Question 6

Refer to Figure Q6 and using Appendix Technical document, type of pipe (refer appendix 4) is copper with a pipe diameter is 25 mm . Water flowing at $70^{\circ} \mathrm{C}$ where the average velocity of the flow is $4 \mathrm{~m} / \mathrm{s}$.
(a) Calculate the Reynolds Number $\left(\mathrm{N}_{\mathrm{R}}\right)$
(b) State the flow either laminar or turbulent flow. Why?
(c) Plot and find the friction factor by using Moody's diagram (refer to Appendices document).
(Note: Detach the Moody's Diagram and attached with your answer sheet)
(10 marks)


Figure Q6: Cooper pipe

## APPENDICES

## Appendix Technical Document



FIGURE 9.2
Moody's diagram. (Source: Pao, R. H. F. 1961. Fluid Mechanics. New York: John Wiley
\& Sons, p. 284)

Table of water properties in SI unit

| table A. 1 <br> SI Units [101 kPa (abs)] |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
| Temperature ( ${ }^{\circ} \mathrm{C}$ ) | Specific Weight $\stackrel{\gamma}{\left(\mathrm{kN} / \mathrm{m}^{3}\right)}$ | $\begin{gathered} \text { Density } \\ \rho \\ \left(\mathbf{k g} / \mathrm{m}^{3}\right) \end{gathered}$ | Dynamic <br> Viscosity <br> $\mu$ <br> (Pa•s) or <br> ( $\mathrm{N} \cdot \mathrm{s} / \mathrm{m}^{2}$ ) | Kinematic Viscosity $\begin{gathered} v \\ \left(m^{2} / \mathrm{s}\right) \end{gathered}$ |
| 0 | 9.81 | 1000 | $1.75 \times 10^{-3}$ | $1.75 \times 10^{-6}$ |
| 5 | 9.81 | 1000 | $1.52 \times 10^{-3}$ | $1.52 \times 10^{-6}$ |
| 10 | 9.81 | 1000 | $1.30 \times 10^{-3}$ | $1.30 \times 10^{-6}$ |
| 15 | 9.81 | 1000 | $1.15 \times 10^{-3}$ | $1.15 \times 10^{-6}$ |
| 20 | 9.79 | 998 | $1.02 \times 10^{-3}$ | $1.02 \times 10^{-6}$ |
| 25 | 9.78 | 997 | $8.91 \times 10^{-4}$ | $8.94 \times 10^{-7}$ |
| 30 | 9.77 | 996 | $8.00 \times 10^{-4}$ | $8.03 \times 10^{-7}$ |
| 35 | 9.75 | 994 | $7.18 \times 10^{-4}$ | $7.22 \times 10^{-7}$ |
| 40 | 9.73 | 992 | $6.51 \times 10^{-4}$ | $6.56 \times 10^{-7}$ |
| 45 | 9.71 | 990 | $5.94 \times 10^{-4}$ | $6.00 \times 10^{-7}$ |
| 50 | 9.69 | 988 | $5.41 \times 10^{-4}$ | $5.48 \times 10^{-7}$ |
| 55 | 9.67 | 986 | $4.98 \times 10^{-4}$ | $5.05 \times 10^{-7}$ |
| $\underline{6}$ | 9.65 | - 984 | $4.60 \times 10^{-4}$ | $4.67 \times 10^{-7}$ |
| 65 | 9.62 | 981 | $4.31 \times 10^{-4}$ | $4.39 \times 10^{-7}$ |
| 70 | 9.59 | 978 | $4.02 \times 10^{-4}$ | $4.11 \times 10^{-7}$ |
| 75 | 9.56 | 975 | $3.73 \times 10^{-4}$ | $3.83 \times 10^{-7}$ |
| 80 | 9.53 | 971 | $3.50 \times 10^{-4}$ | $3.60 \times 10^{-7}$ |
| 85 | 9.50 | 968 | $3.30 \times 10^{-4}$ | $3.41 \times 10^{-7}$ |
| 90 | 9.47 | 965 | $3.11 \times 10^{-4}$ | $3.22 \times 10^{-7}$ |
| 95 | 9.44 | 962 | $2.92 \times 10^{-4}$ | $3.04 \times 10^{-7}$ |
| 100 | 9.40 | 958 | $2.82 \times 10^{-4}$ | $2.94 \times 10^{-7}$ |

Table of water properties in US unit

## TABLE A. 2

U.S. Customary System Units (14.7 psia)

| Temperature ( ${ }^{\circ} \mathrm{F}$ ) | $\begin{gathered} \text { Specific } \\ \text { Weight } \\ \gamma \\ \left(\mathbf{l b} / \mathrm{ft}^{\mathbf{3}}\right) \end{gathered}$ | $\begin{gathered} \text { Density } \\ p \\ \text { (slugs/ } / \mathrm{ft}^{3} \text { ) } \end{gathered}$ | Dynamic <br> Viscosity <br> $\mu$ (lb-s/ft ${ }^{2}$ ) | Kinematic Viscosity $\underset{\left(\mathrm{ft}^{2} / \mathrm{s}\right)}{v}$ |
| :---: | :---: | :---: | :---: | :---: |
| 32 | 62.4 | 1.94 | $3.66 \times 10^{-5}$ | $1.89 \times 10^{-5}$ |
| 40 | 62.4 | 1.94 | $3.23 \times 10^{-5}$ | $1.67 \times 10^{-5}$ |
| 50 | 62.4 | 1.94 | $2.72 \times 10^{-5}$ | $1.40 \times 10^{-5}$ |
| 60 | 62.4 | 1.94 | $2.35 \times 10^{-5}$ | $1.21 \times 10^{-5}$ |
| 70 | 62.3 | 1.94 | $2.04 \times 10^{-5}$ | $1.05 \times 10^{-5}$ |
| 80 | 62.2 | 1.93 | $1.77 \times 10^{-5}$ | $9.15 \times 10^{-6}$ |
| 90 | 62.1 | 1.93 | $1.60 \times 10^{-5}$ | $8.29 \times 10^{-6}$ |
| 100 | 62.0 | 1.93 | $1.42 \times 10^{-5}$ | $7.37 \times 10^{-6}$ |
| 110 | 61.9 | 1.92 | $1.26 \times 10^{-5}$ | $6.55 \times 10^{-6}$ |
| 120 | 61.7 | 1.92 | $1.14 \times 10^{-5}$ | $5.94 \times 10^{-6}$ |
| 130 | 61.5 | 1.91 | $1.05 \times 10^{-5}$ | $5.49 \times 10^{-6}$ |
| 140 | 61.4 | 1.91 | $9.60 \times 10^{-6}$ | $5.03 \times 10^{-6}$ |
| 150 | 61.2 | 1.90 | $8.90 \times 10^{-6}$ | $4.68 \times 10^{-6}$ |
| 160 | 61.0 | 1.90 | $8.30 \times 10^{-6}$ | $4.38 \times 10^{-6}$ |
| 170 | 60.8 | 1.89 | $7.70 \times 10^{-6}$ | $4.07 \times 10^{-6}$ |
| 180 | 60.6 | 1.88 | $7.23 \times 10^{-6}$ | $3.84 \times 10^{-6}$ |
| 190 | 60.4 | 1.88 | $6.80 \times 10^{-6}$ | $3.62 \times 10^{-6}$ |
| 200 | 60.1 | 1.87 | $6.25 \times 10^{-6}$ | $3.35 \times 10^{-6}$ |
| 212 | 59.8 | 1.86 | $5.89 \times 10^{-6}$ | $3.17 \times 10^{-6}$ |



FIGURE 9.1
Pipe wall roughness.

## TABLE 9.1

Pipe roughness-Design values

| Material | Roughness, $\epsilon$ <br> $(\mathbf{m})$ | Roughness, $\epsilon$ <br> $(\mathrm{ft})$ |
| :--- | :---: | :---: |
| Glass, plastic | Smooth | Smooth |
| Copper, brass, lead (tubing) | $1.5 \times 10^{-6}$ | $5 \times 10^{-6}$ |
| Cast iron-uncoated | $2.4 \times 10^{-4}$ | $8 \times 10^{-4}$ |
| Cast iron-asphalt coated | $1.2 \times 10^{-4}$ | $4 \times 10^{-4}$ |
| Commercial steel or welded steel | $4.6 \times 10^{-5}$ | $1.5 \times 10^{-4}$ |
| Wrought iron | $4.6 \times 10^{-5}$ | $1.5 \times 10^{-4}$ |
| Riveted steel | $1.8 \times 10^{-3}$ | $6 \times 10^{-3}$ |
| Concrete | $1.2 \times 10^{-3}$ | $4 \times 10^{-3}$ |

## Formula:

Q $1 \mathrm{in}=0.0254 \mathrm{~m}, 1000 \mathrm{~L}=1 \mathrm{~m}^{3}$

- $P_{a b s}=P_{a t m}+P_{g a g e}$
- $P=\frac{F}{A} \quad$ where $\mathrm{P}=$ Pressure, $\mathrm{F}=$ Force, $\mathrm{A}=\mathrm{C} / \mathrm{S}$ Area

Q $S . G_{s}=\frac{\rho_{s}}{\rho_{\text {water_@_} 4^{\circ} C}}$ where $\mathrm{S} . \mathrm{G}=$ spec.Gravity, $\rho_{\mathrm{s}}=$ density substance
\& $S . G_{s}=\frac{\gamma_{s}}{\gamma_{\text {water_@_4}{ }^{\circ} \mathrm{C}}}$ where $\mathrm{S} . \mathrm{G}=$ spec.Gravity, $\gamma=$ specific weight
$\gamma=\rho g \quad$ where $\rho=$ density, $g=$ gravity
$Q=A v \quad$ where $\mathrm{Q}=$ Volume flow , rate $\mathrm{A}=\mathrm{C} / \mathrm{S}$ area, $v=$ speed
$\rho=\frac{m}{V} \quad$ where $\rho=$ density, $\mathrm{m}=$ mass, $\mathrm{V}=$ =Volume
$M=\rho A v \quad$ where $\mathrm{M}=$ mass flow rate, $\rho=$ density
$W=\rho g V \quad$ where $\mathrm{W}=$ weight, $\mathrm{V}=\mathrm{Volume}$
$W=\gamma Q \quad$ where $\mathrm{W}=$ =weight flow rate, $\gamma=$ specific weight
\& $P=\rho g h \quad$ where $P=$ pressure, $\mathrm{g}=$ gravity, $\mathrm{h}=$ height
Conservation_of_energy: $\frac{p_{1}}{\gamma}+z_{1}+\frac{v_{1}^{2}}{2 g}=\frac{p_{2}}{\gamma}+z_{2}+\frac{v_{2}^{2}}{2 g}$
Q General_energy_equation: $\frac{p_{1}}{\gamma}+z_{1}+\frac{v_{1}^{2}}{2 g}+h_{A}-h_{R}-h_{L}=\frac{p_{2}}{\gamma}+z_{2}+\frac{v_{2}^{2}}{2 g}$
Power : $P_{A}=h_{A} W=h_{A} \nprec ; P_{R}=h_{R} W=h_{R} \gamma Q$
where $\mathrm{P}_{\mathrm{A}}=$ Added Power, $\mathrm{P}_{\mathrm{R}}=$ Removed Power
mechanical_efficiency: $e_{M}=\frac{P_{A}}{P_{i}} ; e_{M}=\frac{P_{o}}{P_{R}}$
where $\mathrm{P}_{\mathrm{i}}=$ power input, $\mathrm{P}_{\mathrm{o}}=$ power output
R Re ynold's_number: $N_{R}=\frac{v D \rho}{\mu}=\frac{v D}{v}$

- $h_{L}=\frac{32 \mu L v}{\gamma D_{2}}=f \times \frac{L}{D} \times \frac{v^{2}}{2 g}$
minor_loss: $h_{L}=\frac{C_{L} v_{1}^{2}}{2 g}$
loss_coefficient : $C_{L}=\left[1-\left(\frac{A_{1}}{A_{2}}\right)^{2}\right]$

