## CONFIDENTIAL

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SET A



# UNIVERSITI KUALA LUMPUR Malaysia France Institute

# FINAL EXAMINATION

# **SEPTEMBER 2014 SESSION**

| SUBJECT CODE    | : | FRD20303                           |
|-----------------|---|------------------------------------|
| SUBJECT TITLE   | : | HEAT TRANSFER & HEAT EXCHANGER     |
| LEVEL           | : | DIPLOMA                            |
| TIME / DURATION | : | 3.00 PM – 6.00 PM<br>( 3.0 HOURS ) |
| DATE            | : | 9 JANUARY 2015                     |

## INSTRUCTIONS TO CANDIDATES

- 1. Please read the instructions given in the question paper CAREFULLY.
- 2. This question paper is printed on both sides of the paper.
- 3. Please write your answers on the answer booklet provided.
- 4. Answer should be written in blue or black ink except for sketching, graphic and illustration.
- 5. This question paper consists of TWO (2) sections. Section A and B. Answer all questions in Section A. For Section B, answer two (2) question only.
- 6. Answer all questions in English.
- 7. Fomulae is appended.

THERE ARE 8 PAGES OF QUESTIONS, EXCLUDING THIS PAGE.

#### **SECTION A (Total: 60 marks)**

INSTRUCTION: Answer ALL questions. Please use the answer booklet provided.

#### **QUESTION 1**

(a) What the definition heat transfer of conduction and convection?

(4 marks)

(b) Between copper and glass, which one has the higher thermal conductivity? Why?

(3 marks)

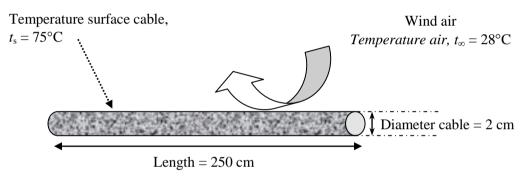


Figure Q1: The Transmission Cable

Referring to figure Q1, The wind at the temperature of 28°C and the speed of blows at 3.5 m/sto a 2 cm diameter transmission cable. The surface temperature of the cable is 75°C and the length of cable is 250 cm. Given that the Nusselt number is 124 and thermal conductivity of air is 0.0281 W/m. °C, determine:-

| (c) | The convective heat transfer coefficient in W/m <sup>2</sup> . <sup>0</sup> C |           |
|-----|---|-----------|
|     |   | (4 marks) |
| (d) | The rate of heat transfer by convection in kW                                 |           |
|     |   | (5 marks) |
| (e) | The amount of heat transfer (kJ) in 2 hours                                   |           |
|     |   | (4 marks) |

#### **QUESTION 2**

(a) What the definition of radiation and Blackbody?

(3 marks)

(b) Give 2 (two) example for both short-wavelength and long-wavelength

(4 marks)

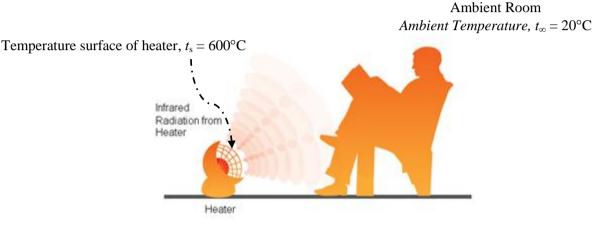


Figure Q2: The Radiant Heater

Referring to figure Q2, a heater use an electricity has an effective 0.87 emissivity at surface temperature of  $600^{\circ}$ C in surroundings temperature at  $20^{\circ}$ C. If the area of the heater surface given is  $0.5m^{2}$  and the Stefan Boltzman coefficient is  $5.676 \times 10^{-8} \text{ W/m}^{2}$ .K<sup>4</sup>, determine:-

(c) The rate of heat transfer by radiation in kW (5 marks)
 (d) The amount of heat rate (kJ) where the period of heat radiant is 3 hours (4 marks)
 (e) The radiant efficiency (in %) of the heater if the input heat by electricity is 20 kW (4 marks)

#### **QUESTION 3**

(a) Write the parameters involved in minimizing heat transmission through the building

(3 marks)

(b) What are relation between Thermal Resistance and U-Value Coefficient?

(2 marks)

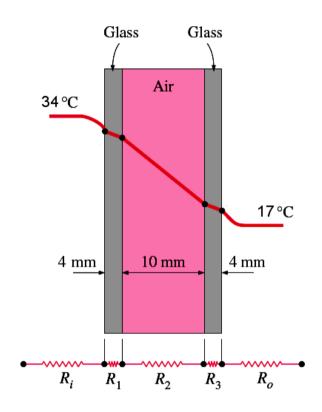


Figure Q3a: The Windows Glass

A1.4 m high and 2.2 m wide double pane windows consists of two layers of glass each and the data of construction for Fig. Q3a as following below:-

Thickness glass = 4 mmThickness stagnant air = 10 mmThermal conductivity of glass=  $0.98 \text{ W/m} \cdot ^{\circ}\text{C}$ Thermal conductivity of air =  $0.026 \text{ W/m} \cdot ^{\circ}\text{C}$ The outside and inside air temperature is  $34 \, ^{\circ}\text{C}$  and  $17 \, ^{\circ}\text{C}$  respectively. The convective heat transfer coefficient on the inner =  $10 \text{ W/m}^2 \cdot ^{^{\circ}\text{C}}$ The convective heat transfer coefficient on the outer =  $25 \text{ W/m}^2 \cdot ^{^{\circ}\text{C}}$ 

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- Determine:-
- The total thermal resistance through the windows, in m<sup>2</sup>.<sup>0</sup>C /W (3 decimal place) (c)
- The U-value coefficient, in W/m<sup>2</sup>. <sup>0</sup>C (d)
- The heat transmission through the window in Watt (e)

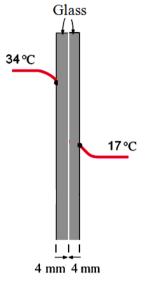


Figure Q3b

The U-value coefficient (W/m<sup>2</sup>. <sup>0</sup>C) for two glasses only (without air gap) as shown in (f) figure Q3b.

(4 marks)

What is the conclusion that can be made between figure Q3a and Q3b? (g)

(2 marks)

(3 marks)

(3 marks)

(3 marks)

(6 marks)

#### SECTION B (Total: 40 marks)

INSTRUCTION: Answer only TWO questions.

Please use the answer booklet provided.

### **Question4**

- (a) Describe and sketch the temperature profile across the two tubes for:
  - i. Parallel Flow heat exchanger operation and,
  - ii. Counter Flow heat exchanger operation

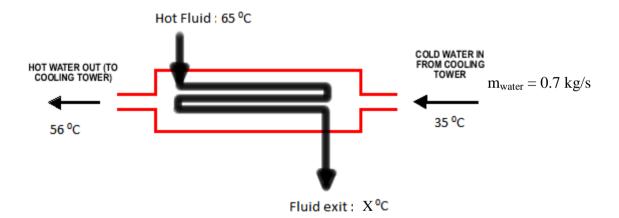


Figure Q4: The heat exchanger of Condenser

Referring to Figure Q4, hot fluid at a temperature of  $65^{\circ}$ C (mC<sub>p</sub>=4.5 kW/K) enters through an outer tube and exchanges counter flow heat with the shell having water from cooling tower enters at 35 °C and exits at 56°C with the mass flow rate of water is 0.7 kg/s and specific heat of water is 4.18 kJ/kg. °C, determine:

| (b) | Prove that the capacity of energy balance is 61.5 kW    |           |
|-----|---|-----------|
| (-) | The suit terms excluse of the bet water is $^{0}$       | (3 marks) |
| (c) | The exit temperature of the hot water in <sup>0</sup> C | (4 marks) |
| (d) | Sketch the temperature profile.                         | 、 ,<br>,  |
| ( ) |   | (3 marks) |
| (e) | The Log Mean Temperature Different, ∆LMTD               | (4 marks) |

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### **Question 5**

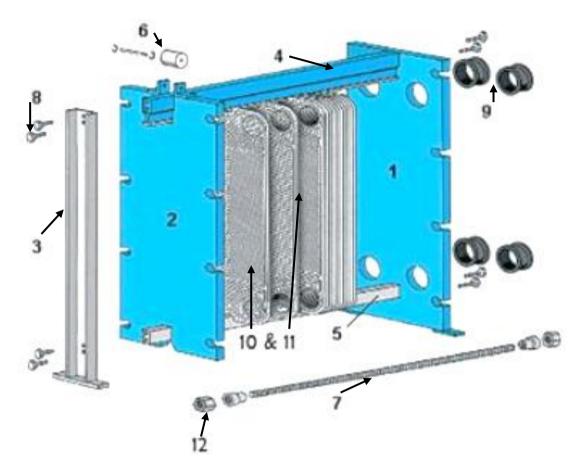


Figure Q5a: Plate heat exchanger

(a) By referring to figure Q4a, Insert the components number in the blank table Q4a below:-*(Please rewrite table Q5a into answer sheet)* 

Table Q5a: The Component of Plate Heat Exchanger

| Top Carrying Beam | Fixed Plate       | Roll Assembly     |
|-------------------|-------------------|-------------------|
| Support Bar       | Lower Guiding Bar | Liners            |
| Tightening Rods   | Gasket            | Centering Bushing |
| Pressure Plate    | Plate             | Fixing Bolts      |

(6 marks)

Below is a parallel flow heat exchanger made of mild steel specification as shown in Figure Q5b below.

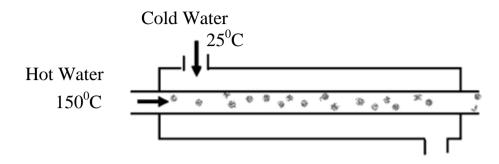


Figure Q5b: Basic Shell-And-Tube heat exchanger

## DATA

Warm water enters at 25<sup>o</sup>C with the mass flow rate, m = 2 kg/s Hot water enters at 150<sup>o</sup>C; with the mass flow rate, m = 8 kg/s Heat Exchanger Surface Area, A = 45 m<sup>2</sup>, Overall heat transfer coefficient selection, U: 464.4 W/m<sup>2</sup>.K Specific heat for water , Cp = 4.18 kJ/kg.K

Calculate:-

| (b) | The Capacitance rate of $C_{max}$ and $C_{min}$                          |   |
|-----|--|---|
|     |  | (4 marks)                               |
| (C) | The Capacitance Ratio, C   |   |
|     |  | (2 marks)                               |
| (d) | The Number of Transfer Unit  |   |
|     |  | (2 marks)                               |
| (e) | The value of effectiveness heat exchanger by sketching in Appendix 1 (Ap | pendix 1                                |
|     | need to be returned with the answer sheet),                              |   |
|     |  | (3 marks)                               |
| (f) | The rate of heat exchanger, in kW  |   |
|     |  | (3 marks)                               |
| (f) |  | ζ , , , , , , , , , , , , , , , , , , , |

#### **Question 6**

(a) What is function of Transverse baffles?

(2 marks)

(b) Write 1 (one) advantage and disadvantage for shell and tube heat exchanger

(3 marks)

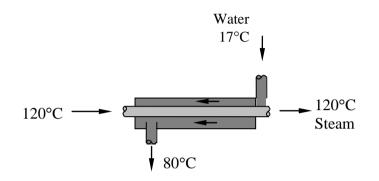


Figure Q6: Shell & Tube Heat Exchanger

Water (Cp = 4.18kJ/kg.<sup>0</sup>C) enters the 2.5cm internal diameter tube of double pipe counter flow at a temperature of 17 <sup>o</sup>C and the mass flow rate of 0.5 kg/s heat exchanger as shown in figure Q6. The water is heated by condensing the steam at 120 <sup>o</sup>C. If the total coefficient of exchange is 0.9 kW/m<sup>2</sup>.<sup>o</sup>C, and the water is heated to 80 <sup>o</sup>C, Calculate,

| (c) | The heat transfer rate of the system, in kW (energy balance) |           |
|-----|--|-----------|
|     |  | (4 marks) |
| (d) | The Log mean temperature Different, LMTD                     |           |
|     |  | (5 marks) |
| (e) | The surface area in m <sup>2</sup>                           |           |
|     |  | (3 marks) |
| (f) | The pipe length of heat exchanger in meter                   |           |
|     |  | (3 marks) |

#### END OF QUESTION

#### ID NO: \_\_\_\_

## **APPENDIX 1 (TO BE RETURNED)**

### The effectiveness of parallel and counter-flow heat exchangers from CHART

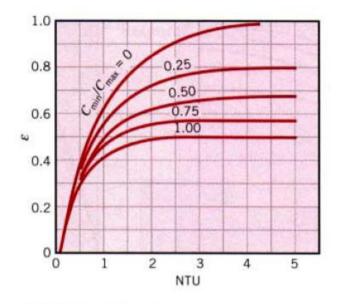
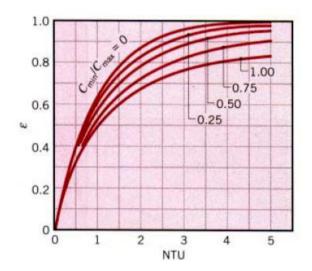
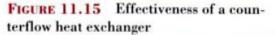


FIGURE 11.14 Effectiveness of a parallel-flow heat exchanger





**APPENDIX 2** 

#### 1. Fourier's Law of Conduction.

$$Q = k \times A \times \frac{\Delta t}{\Delta x}$$

Where

Q = The rate of heat transfer by conduction, W or Btu/hr k = The thermal conductivity,  $\frac{W}{m.c}$  or  $\frac{Btu}{hr.ft.F}$ A = the area surface to the direction of heat flow, m<sup>2</sup> or ft<sup>2</sup>  $\Delta t$  = different temperature (t<sub>higher</sub> - t<sub>lower</sub>), unit <sup>0</sup>C or <sup>0</sup>F  $\Delta x$  = thickness distance in the direction of heat flow, m or ft

#### 2. Forced Convection - NEWTON'S LAW

$$Q = h \times A \times (t_{surf} - t_{surr})$$

Q = The rate of heat transfer by convection, W or Btu/hr h = The convection heat-transfer coefficient,  $\frac{W}{m^2.c}$  or  $\frac{Btu}{hr.ft^2.F}$  A = exposed surface area, m<sup>2</sup> or ft<sup>2</sup>  $t_s =$  temperature of the immersed object (surface), <sup>0</sup>C or F  $t_0 =$  temperature of the fluid surrounding, <sup>0</sup>C or F

### 3 Stefan Boltzman Law's

$$Q = \varepsilon \times A \times \sigma \times (T_1^4 - T_2^4)$$

Where:

 $\begin{array}{l} Q = \text{The heat transfer rate by radiation, Watt or Btu/hr} \\ \sigma = Stefan-Boltzman constant:-0.1714 \times 10^{-8} \frac{Btu}{hr.ft^2.R^4} \ \text{ for IP unit} \\ 5.676 \times 10^{-8} \frac{W}{m^2.K^4} \ \text{ for SI unit} \\ \text{A} = \text{Area surface in m}^2 \text{ or ft}^2 \\ \epsilon = \text{emissivity (Blackbody = 1 or Greybodies } 0 < \epsilon < 1) \\ \text{T}^4 = (\text{T}_1^{-4} - \text{T}_2^{-4}), \quad \text{T}_1 = \text{Temperature of the first body, K or } {}^0\text{R} \\ \text{T}_2 = \text{Temperature of second body, K or } {}^0\text{R} \end{array}$ 

#### 4. Heat transmission,

$$Q_{trans} = Area \times uvalue \times (t_{out} - t_{in})$$

Where, A = Area of the wall surface, 
$$\mathbf{m}^2 \operatorname{or} \mathbf{ft}^2$$
  
U = U-Value,  $\frac{W}{m^2.C}$  or  $\frac{Btu}{hr.ft^2.F}$   
 $t_{out}$  = Temperature out,  ${}^{\mathrm{o}}\mathbf{C} \operatorname{or} {}^{\mathrm{o}}\mathbf{F}$   
 $t_{in}$  = Temperature in,  ${}^{\mathrm{o}}\mathbf{C} \operatorname{or} {}^{\mathrm{o}}\mathbf{F}$   
 $Uvalue = \frac{1}{TOTAL THERMAL RESISTANCE}$   
 $Uvalue = \frac{1}{\Sigma R_T}$ 

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#### a) **R** = Thermal Resistance.

$$R = \frac{\Delta x}{k}$$

**b)** C = Thermal Conductance.  $C = \frac{1}{R}$ , Resistance due conductance,  $R = \frac{1}{c}$  **c)** = Surface conductance of convective heat transfer  $R_{out} = \frac{1}{h_{out}}$ ,  $R_{out} = \frac{1}{h_{out}}$ 

5. Log mean temperature different, ΔLMTD

$$Q = U \times A \times \Delta LMTD$$

where

Q = heat transfer rate, W or Btu/hr

 $U = \text{overall heat transfer coefficient}, \frac{m^2 C}{W}$  or  $\frac{hr.F.ft^2}{Btu}$  $A = \text{Area Surface, m}^2 \text{ or ft}^2$ 

 $\Delta LMTD = \log$  mean temperature difference, K or R

$$\Delta LMTD = \frac{(\Delta T_2 - \Delta T_1)}{Ln\left(\frac{\Delta T_2}{\Delta T_1}\right)}$$

6. Energy Balance.

 $Q_A = Q_B = m \times C_p \times \Delta t$ 

Where

Q = the rate of capacity, W m = mass of fluid, kg/s Cp = specific heat, J/kg.C or J/kg.K  $\Delta T$  = Different temperature, K

#### 7. NTU Method

$$Q = \varepsilon \times C_{min} \times t_{hot in} - t_{cold in}$$

where  $\varepsilon = effectiveness$ 

 $C_{min}$  = Heat capacity rate.  $t_{hot in}$  = temperature hot inlet  $t_{cold in}$  = Temperature cold inlet

a) The capacitance rate

 $C_{hotfluid} = m_{hotfluid} \times c_{p_{hotfluid}}$  $C_{coldfluid} = m_{coldfluid} \times c_{p_{coldfluid}}$  $C_{h} \text{ and } C_{c} = \text{ The capacitance rate, W/K}$ m = The mass flow rate, kg/s $C_{p} = \text{ The Specific heat, J/kg.K or J/kg.C}$ 

Note:-

 $C_{coldfluid}$  or  $C_{hotfluid}$  is lower value =  $C_{min}$  $C_{coldfluid}$  or  $C_{hotfluid}$  is higher value =  $C_{max}$ 

b) The Capacitance Ratio, Cr as following below:-

$$C_r = \frac{C_{min}}{C_{max}}$$

c) The Variable formula of NTU

$$NTU = \frac{U \times A}{C_{min}}$$

where , NTU = number of transfer unit in dimensionless parameter

U = Overall heat Transfer Coefficient

A = Area surface

 $C_{\min}$  = depending the value of  $C_h$  and  $C_c$ 

#### 8. Radiant Efficiency

Radiant Efficiency, 
$$\eta$$
 rad =  $\frac{Output heat by radiation}{input heat by natural gas} \times 100\%$